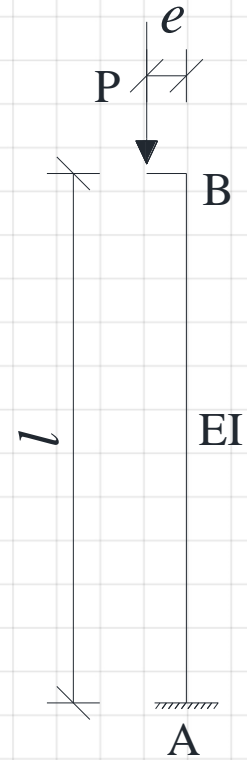
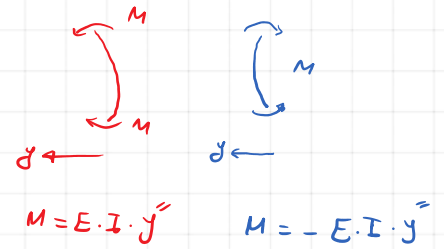
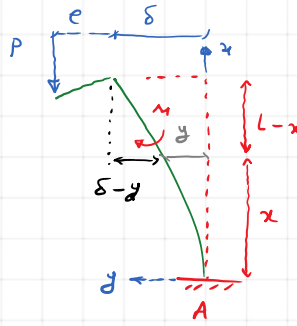
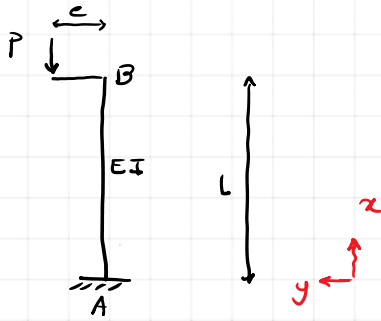


In a cantilever column, the compressive load is applied by the eccentricity of  $e$ , as shown in the figure. The column height is  $l$  with the bending rigidity of  $EI$ .

- Derive the maximum horizontal displacement of point B as a function of eccentricity, force  $p$ , and  $EI$ .
- Sketch the relation between the maximum displacement and the applied force  $p$ <sup>1</sup>.
- Determine the maximum bending moment and maximum normal stress.



<sup>1</sup> The best solution would be using the non-dimensional variables



$$M = P(e + \delta - y) = P(e + \delta) - P \cdot y$$

$$EI \cdot y'' = P(e + \delta) - P \cdot y$$

$$EI \cdot y'' + P \cdot y = P \cdot (e + \delta)$$

$$y'' + \frac{P}{EI} \cdot y = \frac{P}{EI} (e + \delta) \quad \frac{P}{EI} = \lambda^2$$

$$y'' + \lambda^2 y = \lambda^2 (e + \delta) \Rightarrow$$

$$y = A \cdot \sin \lambda x + B \cos \lambda x + (e + \delta), \quad y' = A \cdot \lambda \cdot \cos \lambda x - B \cdot \lambda \cdot \sin \lambda x$$

$$\begin{cases} y(0) = 0 \rightarrow B + e + \delta = 0 \Rightarrow B = -e - \delta \\ y'(0) = 0 \rightarrow A \cdot \lambda = 0 \Rightarrow A = 0 \end{cases}$$

$$y = -(e + \delta) \cos \lambda x + (e + \delta) = (e + \delta) [1 - \cos \lambda x]$$

$$y = (e + \delta) [1 - \cos \lambda x]$$

$$y(L) = \delta \Rightarrow \delta = (e + \delta) [1 - \cos \lambda L]$$

$$\cancel{\delta} = e + \cancel{\delta} - (e + \delta) \cos \lambda L \Rightarrow e - e \cos \lambda L - \delta \cdot \cos \lambda L = 0 \Rightarrow$$

$$\delta \cos \lambda L = e - e \cos \lambda L \Rightarrow \delta = \frac{e(1 - \cos \lambda L)}{\cos \lambda L} = e \left( \frac{1}{\cos \lambda L} - 1 \right)$$

$$\rightarrow \boxed{\delta = e (\sec \lambda L - 1)}$$

$$\delta = e (\sec \lambda L - 1)$$

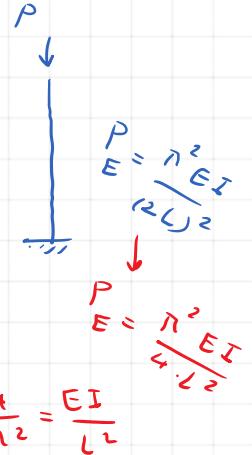
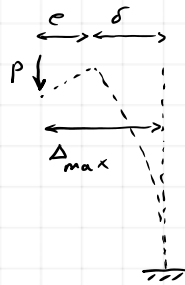
$$\Delta_{max} = \delta + e = e \sec \lambda L - e + e$$

$$\Delta_{max} = e \cdot \sec \lambda L$$

$$e = \alpha \cdot L$$

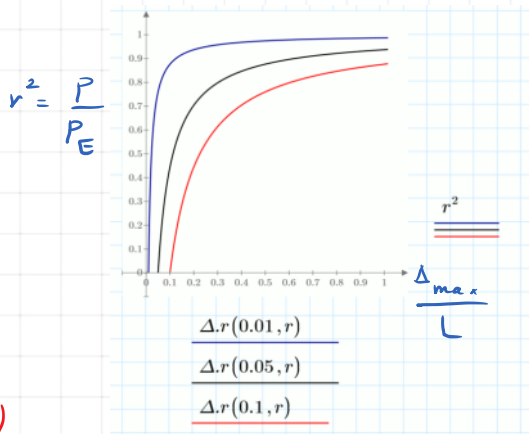
$$\lambda^2 = \frac{P}{EI} \Rightarrow \lambda^2 \cdot L^2 = \frac{PL^2}{EI} = \frac{P}{EI/L^2} = \frac{P}{\frac{P_E \cdot 4}{\pi^2}} = \frac{\pi^2}{4} \left( \frac{P}{P_E} \right)$$

$$\lambda^2 \cdot L^2 = \frac{\pi^2}{4} \cdot r^2 \Rightarrow \lambda L = \frac{\pi}{2} \cdot r$$



$$\Delta_{max} = \alpha \cdot L \cdot \sec \left( \frac{\pi}{2} \cdot r \right)$$

$$\frac{\Delta_{max}}{L} = \alpha \cdot \sec \left( \frac{\pi}{2} \cdot r \right)$$

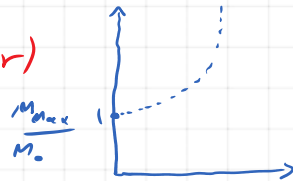


$$M_{max} = P \cdot \Delta_{max} = P \cdot \alpha \cdot L \cdot \sec \left( \frac{\pi}{2} \cdot r \right)$$



$$M_{max} = P \cdot e \cdot \sec \left( \frac{\pi}{2} \cdot r \right)$$

$$\frac{M_{max}}{M_0} = \sec \left( \frac{\pi}{2} \cdot r \right)$$



$$\sigma = \frac{P}{A} + \frac{M}{W}$$

$$\sigma_{max} = \frac{P}{A} + \frac{P \cdot e \cdot \sec \left( \frac{\pi}{2} \sqrt{\frac{P}{P_E}} \right)}{W}$$

$$\sigma_{max} = P \left[ \frac{1}{A} + \frac{e \cdot \sec \left( \frac{\pi}{2} \sqrt{\frac{P}{P_E}} \right)}{W = \frac{I}{C}} \right]$$